



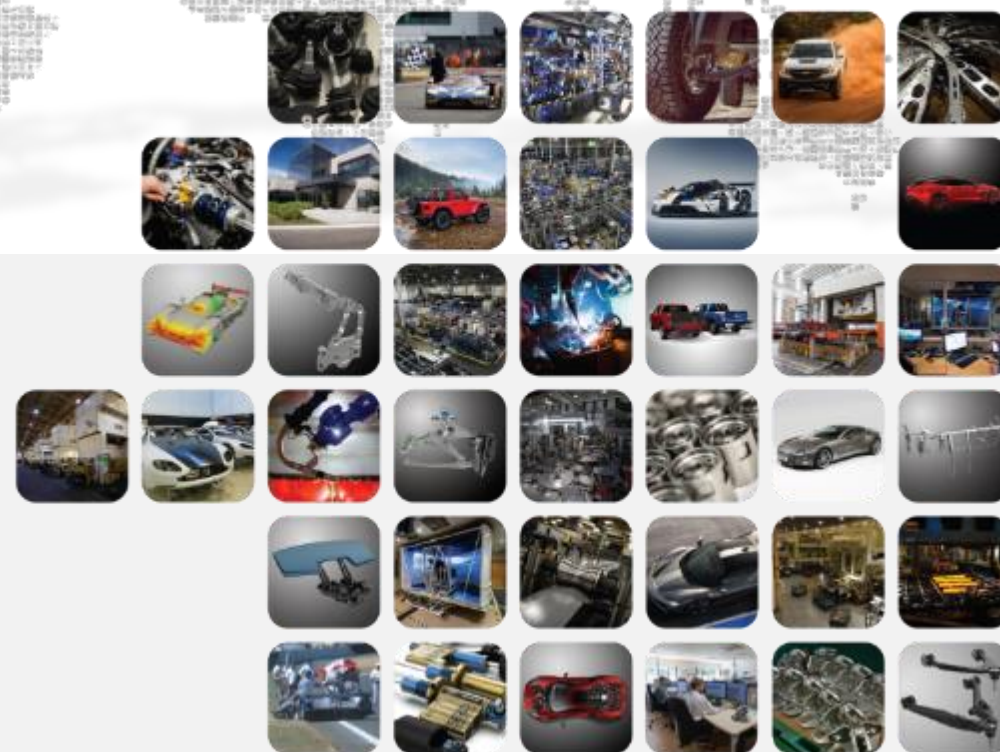
Enhancing Virtual Ride Quality Subjective Assessment Using FTire

2024 Zero Prototypes Summit

May 15th-16th, 2024 – Udine, Italy

Presented by:

Tony Yardley, Multimatic
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Acknowledgements



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Multimatic



Industry Leaders in Tire Design, Analysis, and Manufacturing



Globally recognized expertise in tire modeling
Michelin Developed Models
Commercial Tire Models

cosin
scientific software



MFSwift

MFTyre

CDTire

Multimatic – The Company



Multimatic is a supplier that delivers manufactured products and engineering services to the global automotive industry at any volume level. Headquartered in Toronto, Canada with facilities in North America, Europe and Asia.

Low Volume Production

High Volume Production

Suspension Products



Structural Products



Mechanism Products

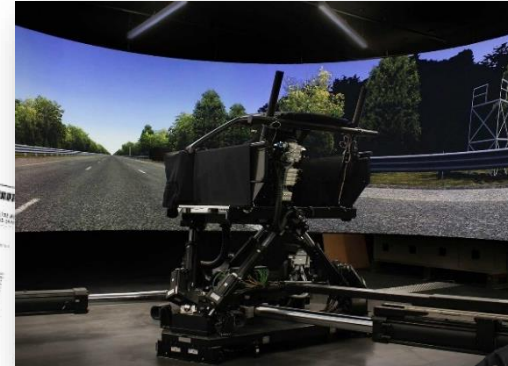
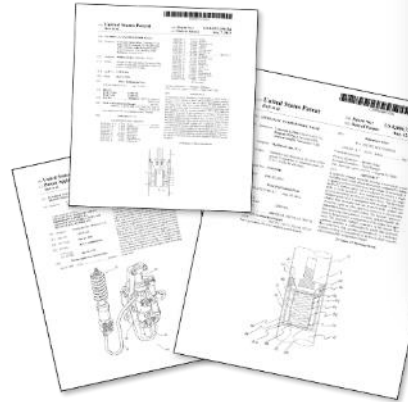


Multimatic Engineering

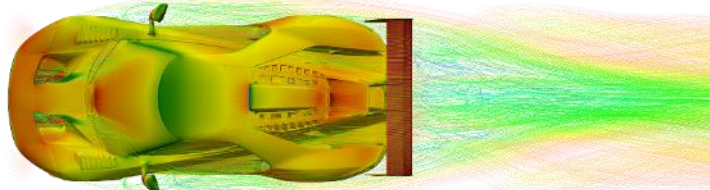


Multimatic Engineering delivers integrated design, analysis, prototyping, testing, validation and R&D activities for products ranging from individual components to complete vehicles

Research and Development

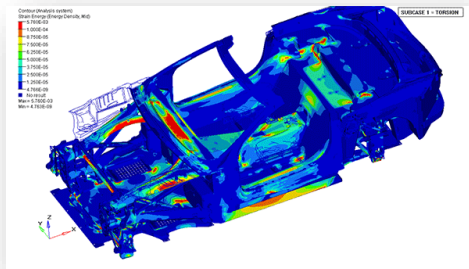


Vehicle Dynamics

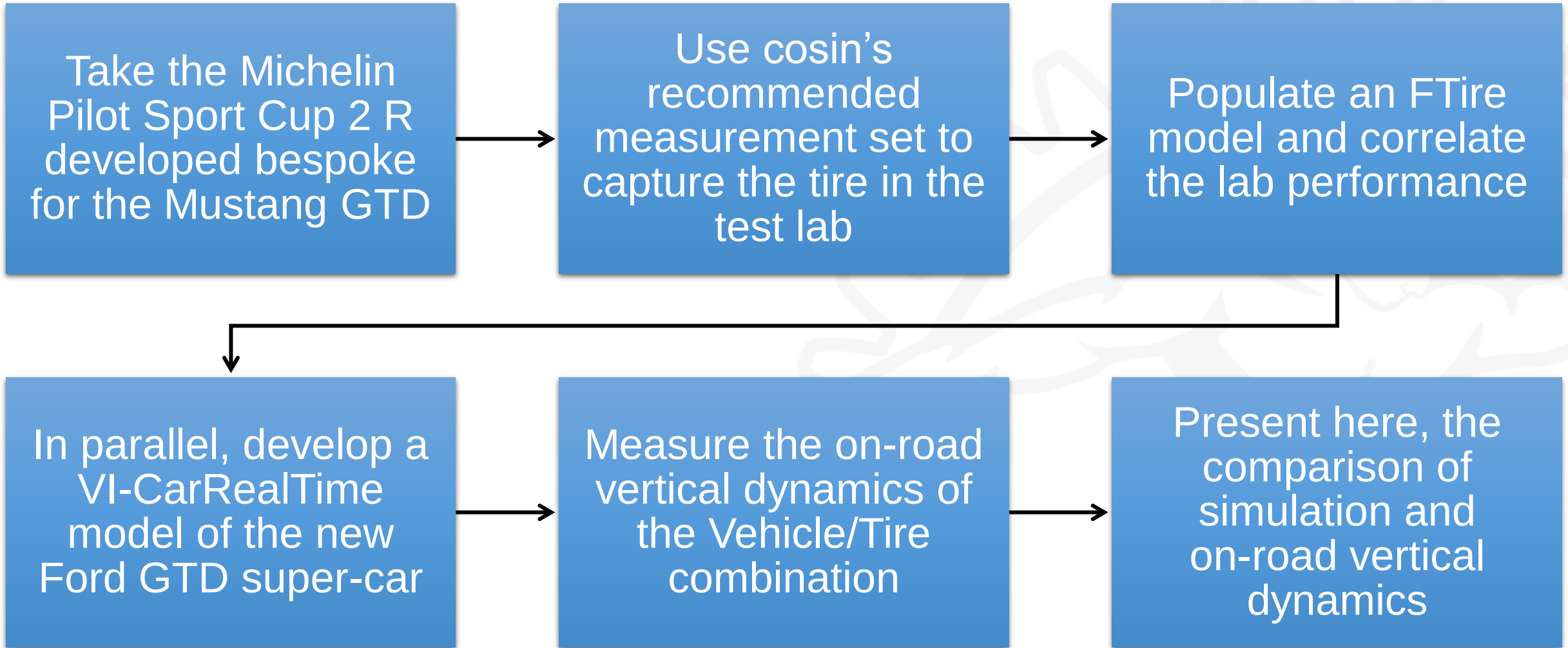


Advanced simulation

Component and system engineering



Project Scope



Tire Simulation Models

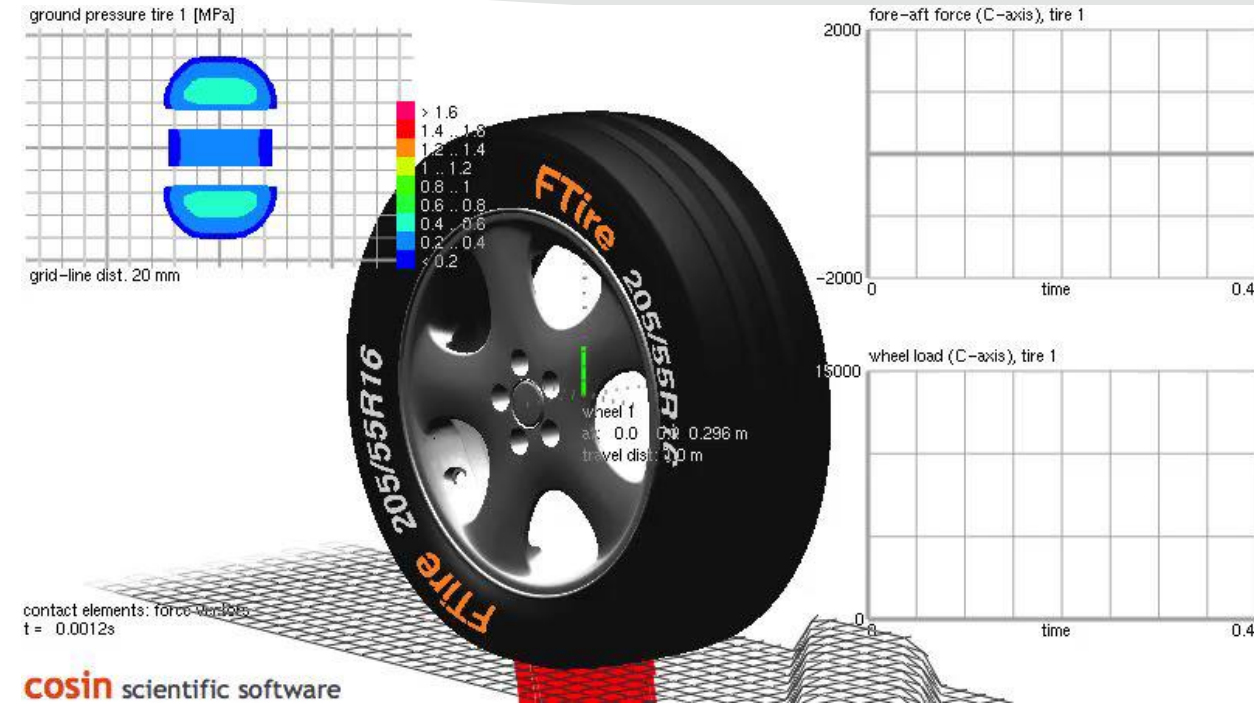
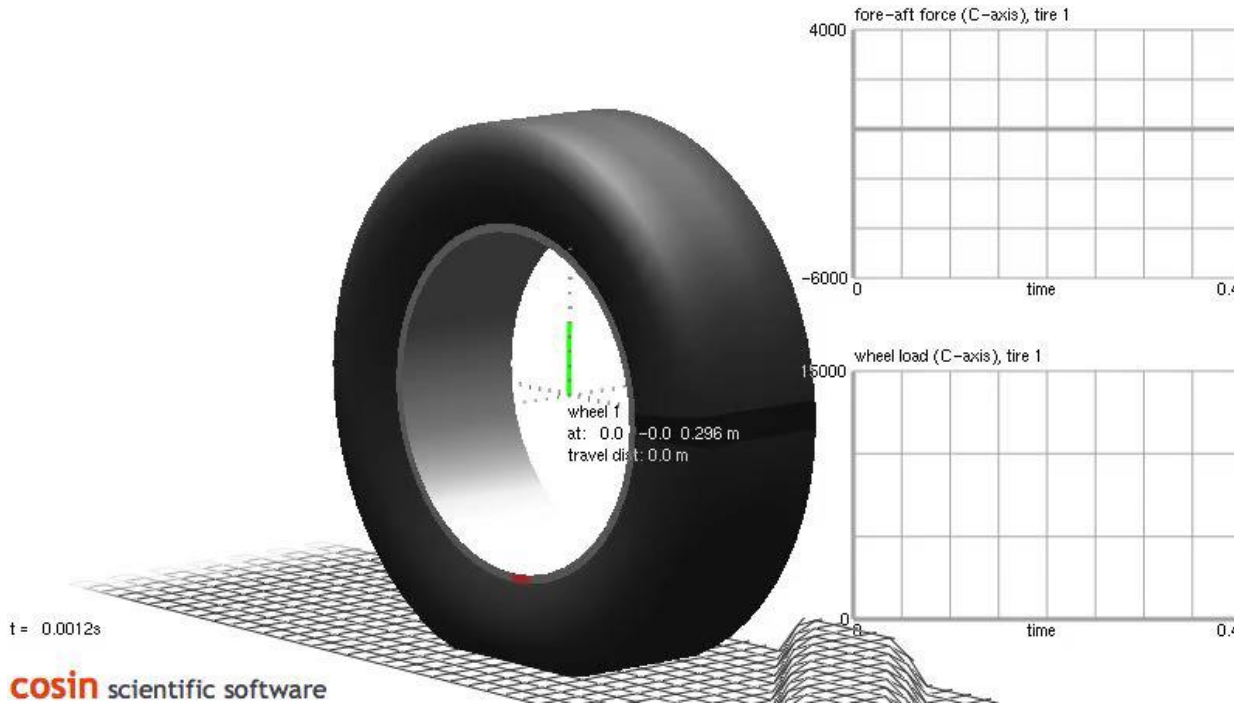


Approximation (MF) Models

one single
contact point

Physical Model - FTire

many hundreds
contact points,



Simulating Real Tire Dynamics



Core mechanical model:

1. Tire structure

consisting of bead, carcass, belt: sub-divided into 60 .. 360 (or more) belt segments
connected to each other and to the rim by

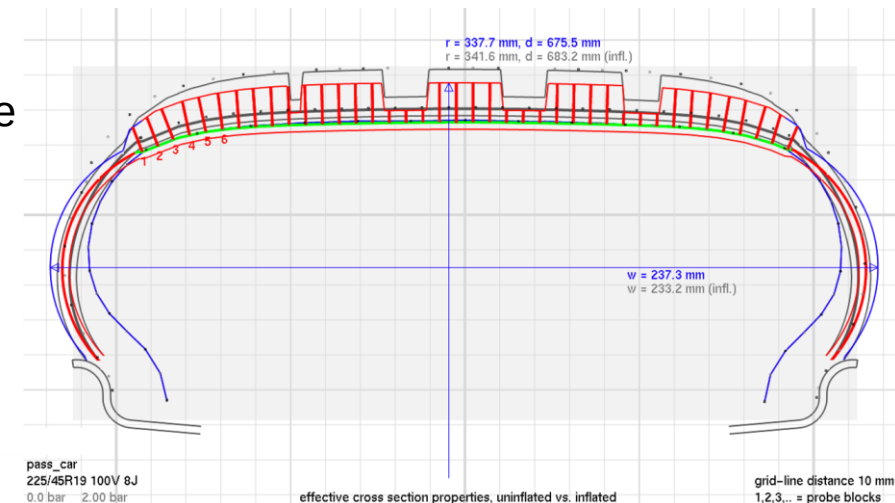
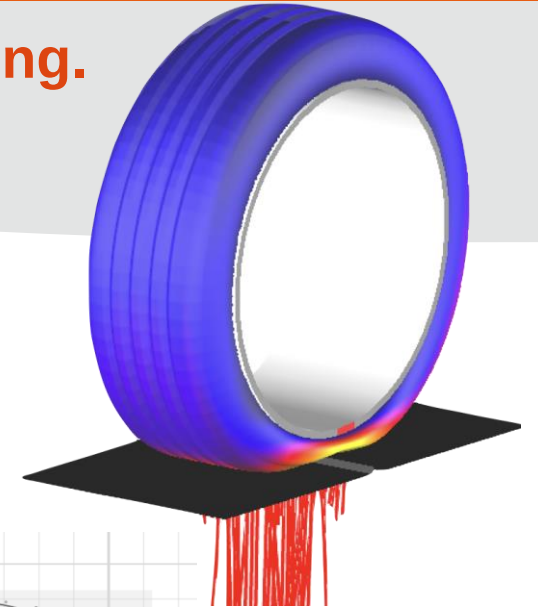
- non-linear stiffness and damping
- pressure-dependent
- distributed stiffness and damping elements

2. Tread

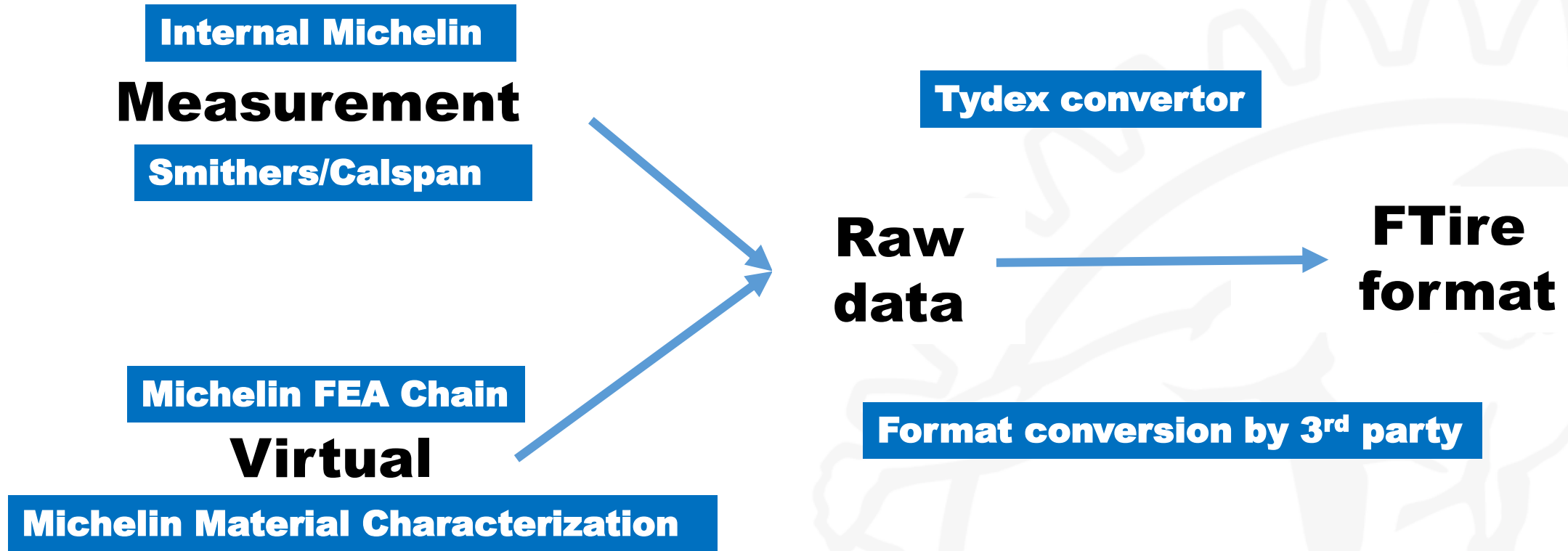
consisting of 1000 .. 10000 (or more) contact elements
connected to belt segments, sticking / sliding on road surface

- non-linear non-linear rubber stiffness and damping
- friction coefficient dependent on
 - sliding velocity
 - ground pressure
 - temperature

Structural Dynamics Modelling.



Data-generation workflow



Measurements/Results



Mass & Inertia

Tire mass

Tire axial moment of inertia I_{yy} (If not specified, will be estimated)

Rim I_{yy}

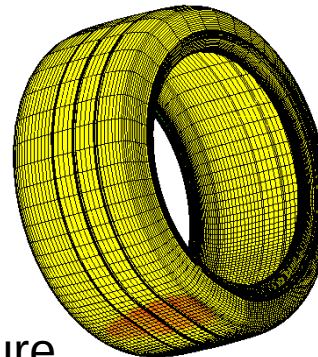
Tire polar moment of inertia I_{xx}/I_{zz} (Optional)

Rim I_{xx}/I_{zz} (Optional)

Geometry

Mounted/Inflated profile

Footprints



Rmax

Unloaded radius v.s. inflation pressure

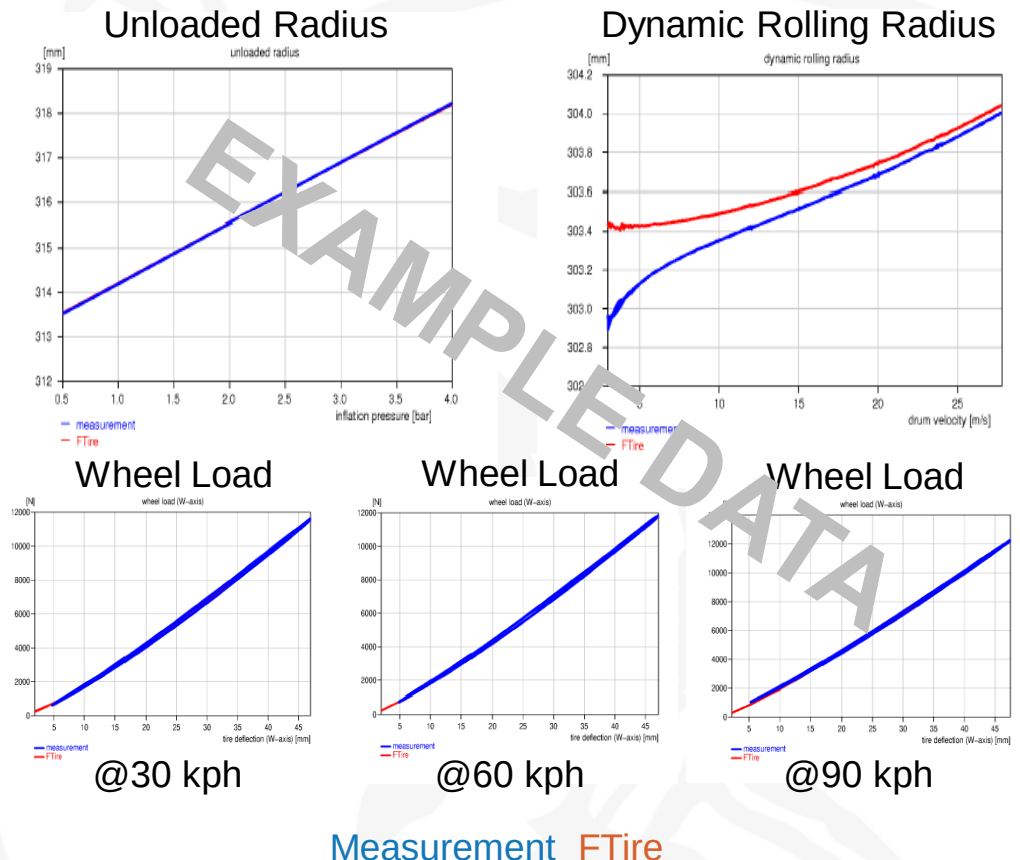
Rdyn

load fixed (50% LI), slowly apply speed variation

Vdyn

Speed fixed, apply sinusoidal load condition

Measure curves at several speed



Measurements/Results



Vertical Stiffness

On flat surface

With/Without camber

On cleat

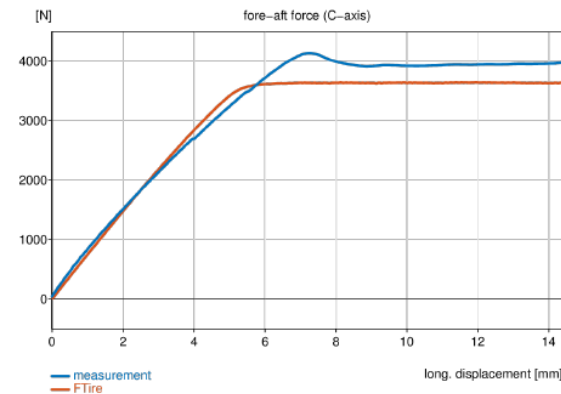
Oblique

Pure lateral slip & Pure longitudinal slip (no combined slip)

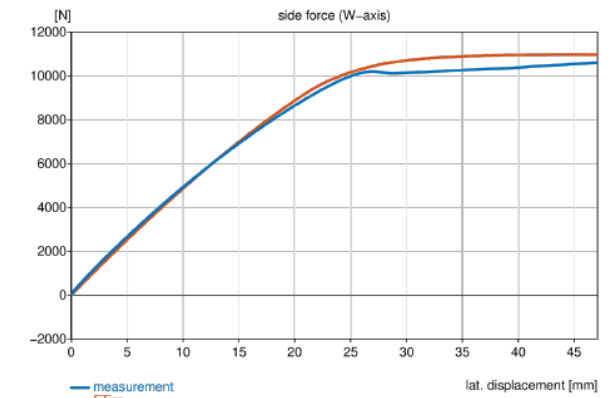
3 load conditions - stationary
Pure lateral slip: no camber/ positive camber / negative camber

Note that standard format requires 6 forces & moments signal; actual signal used is F_x , F_y , F_z and M_z .

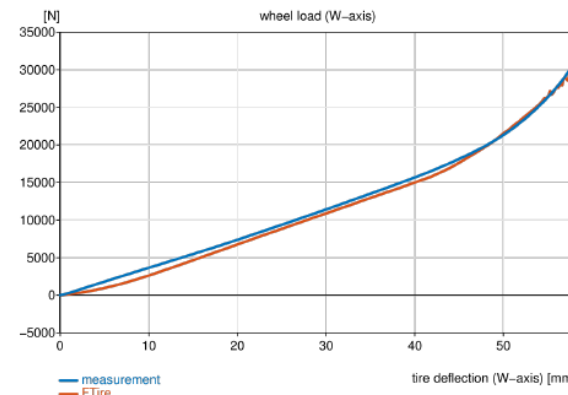
Longitudinal Stiffness



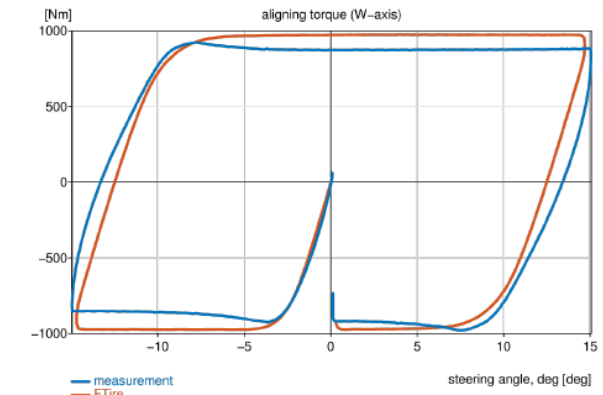
Lateral Stiffness



Bottoming: "pinching"



Torsional Stiffness



Measurement FTire

Measurements/Results



Dynamic cleat test

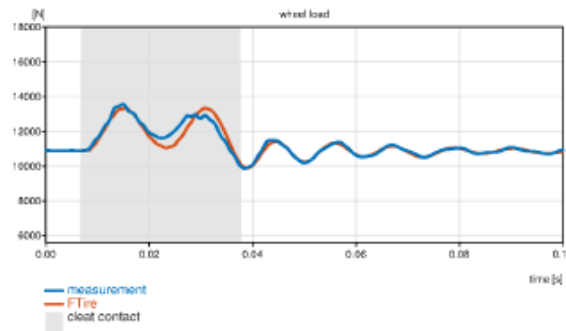
Transversal / oblique cleat

10x20 mm unless special requirement by customer → consider lab capacity!

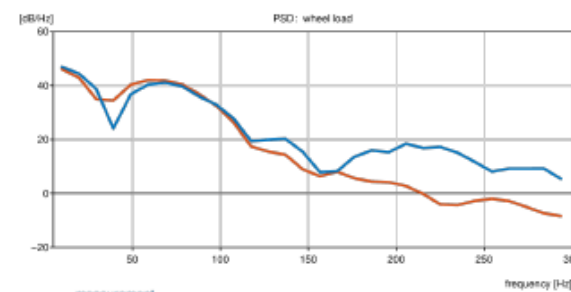
Standard condition: 3 load X 3 speed (30, 60, 90 kph)

→ At 2kph or other low speed is not required in standard condition

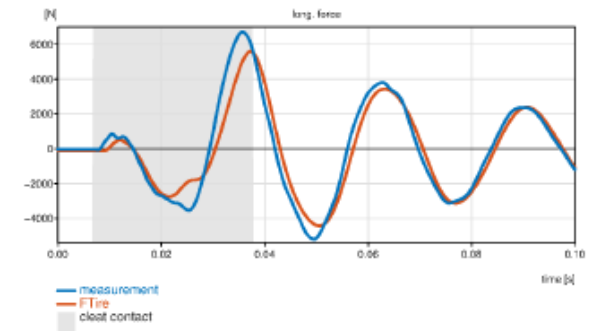
Wheel Load



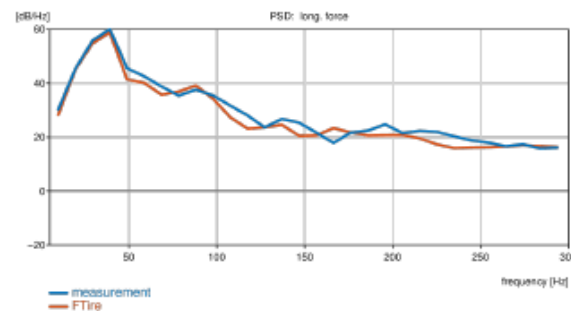
Wheel Load PSD



Longitudinal Force



Longitudinal Force PSD



Measurement

FTire

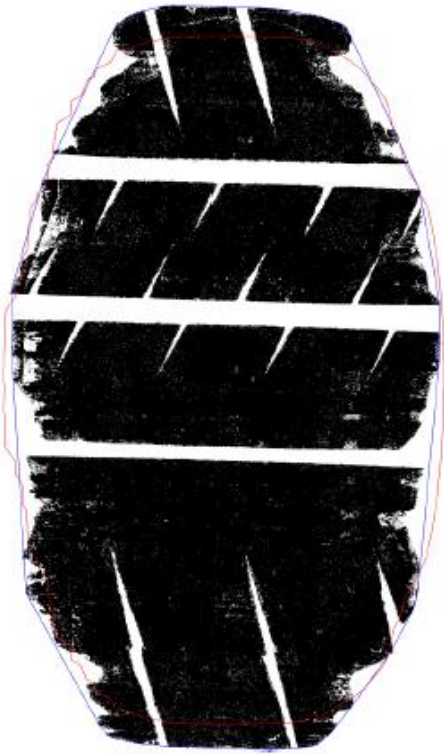
Measurements/Results



Contact Patch Model

Good correlation for static tire with limited camber

Fitting becomes more challenging with camber and speed, especially at lower loads

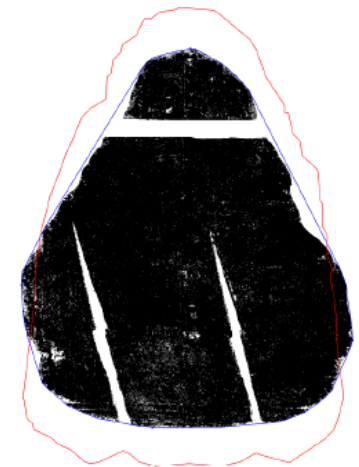


$F_z = 11.18 \text{ kN}$, Camber Angle = 0.00 deg, 2.3 bar



$F_z = 11.18 \text{ kN}$, Camber Angle = 6.00 deg, 2.3 bar

Measurement
FTire



$F_z = 3.73 \text{ kN}$, Camber Angle = 0.00 deg, 2.3 bar

Measurements/Results



Side Force Generation / Aligning Moment & Fore-Aft Force for Rolling Tire

Side force generation as a function of slip can be fit, but a trade off exists for maintaining frequency response and NVH characteristics with an accurate aligning moment

Oscillation in the model around high braking forces (right hand plot)

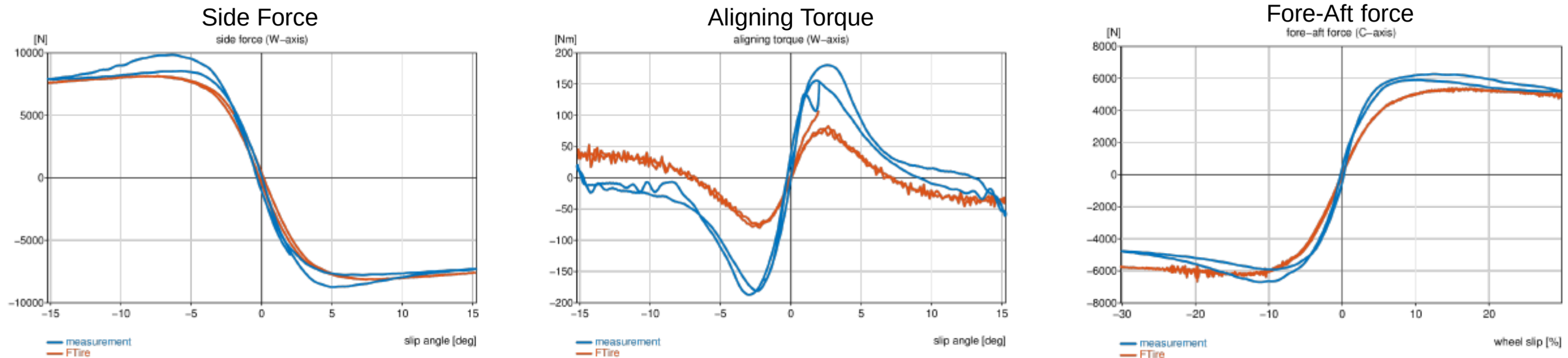


Fig. 10.22: slip angle sweep (drum diam. 9999 m, 59.85 km/h), $F_z=5.58$ kN, 2.3 bar, camber angle=0 deg

Measurement FTire

Full Vehicle Testing



Another of Multimatic's Niche vehicles ...

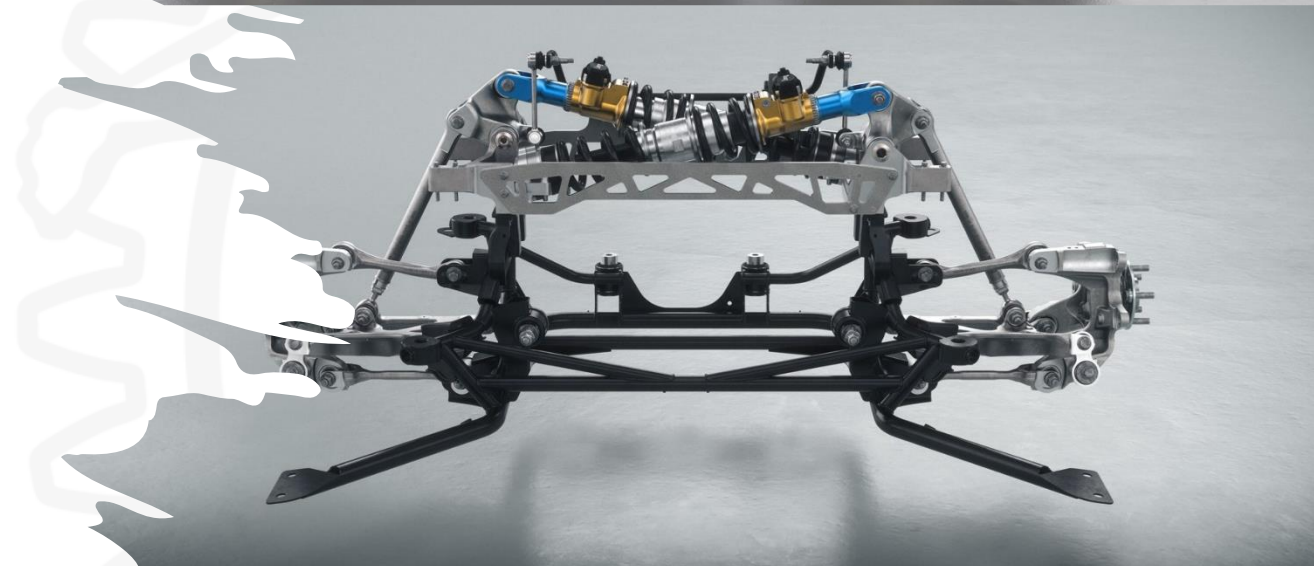
Ford Mustang GTD Prototype

In-board rear suspension

Multimatic's ASV (Adaptive Spool Valve),
DHR (Dual - Height Rate)

Front Tire 325/30R20

Rear Tire 345/30R20



Tires



MICHELIN

PILOT SPORT CUP 2 R

BESPOKE DEVELOPMENT TIRES FOR THE MUSTANG GTD

FRONT: 325/30ZR20

REAR: 345/30ZR20

Exceptional track performance made to last.

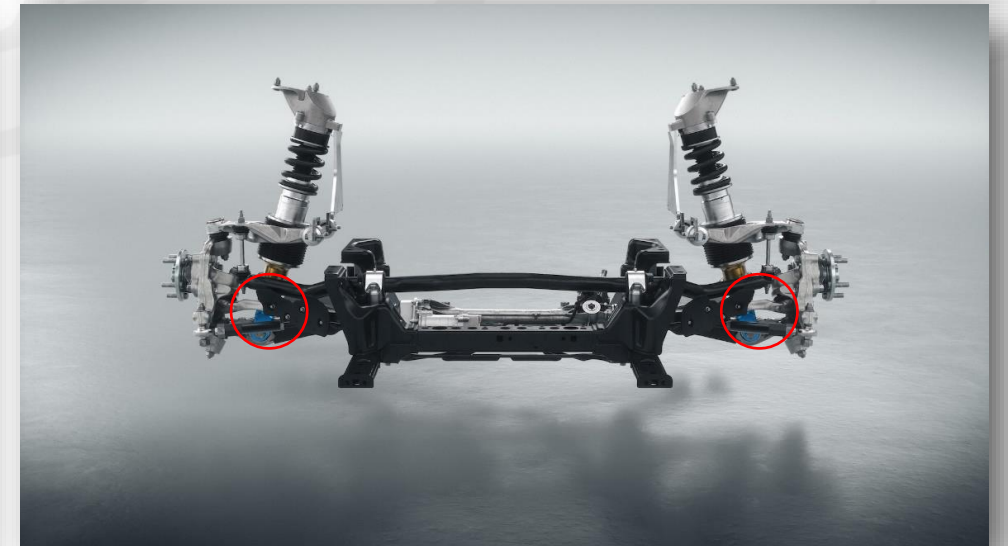
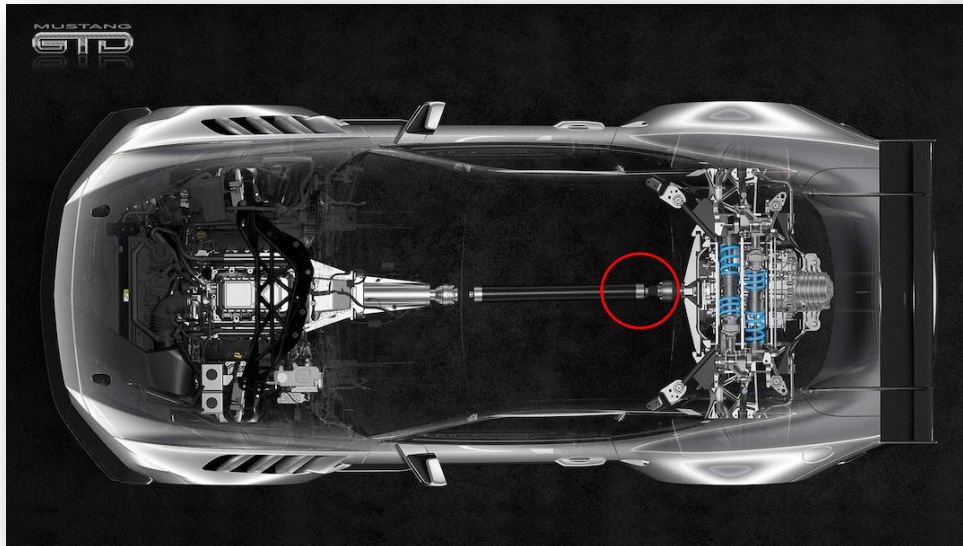
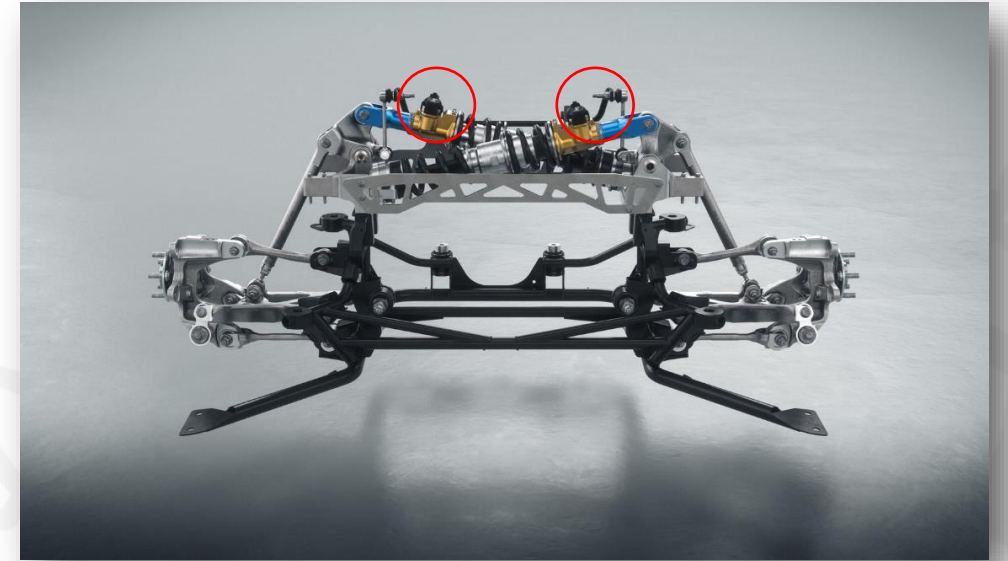
MICHELIN Pilot Sport Cup 2 R track tires are designed for ultra-high performance for sports and hypersport cars, with sensational times.

- MICHELIN Semi-Slick high performance track tires, road approved
- Developed to be a Nürburgring laptime weapon
- In the dry, extreme lateral grip and driving precision
- Virtual sizing study along with multiple design evolution loops in partnership with Multimatic Mustang GTD Development

Logging Setup

4x 1000 Hz 3 axis accelerometers mounted on the damper

1x 200 Hz 3 axis accelerometer & 3 axis gyro

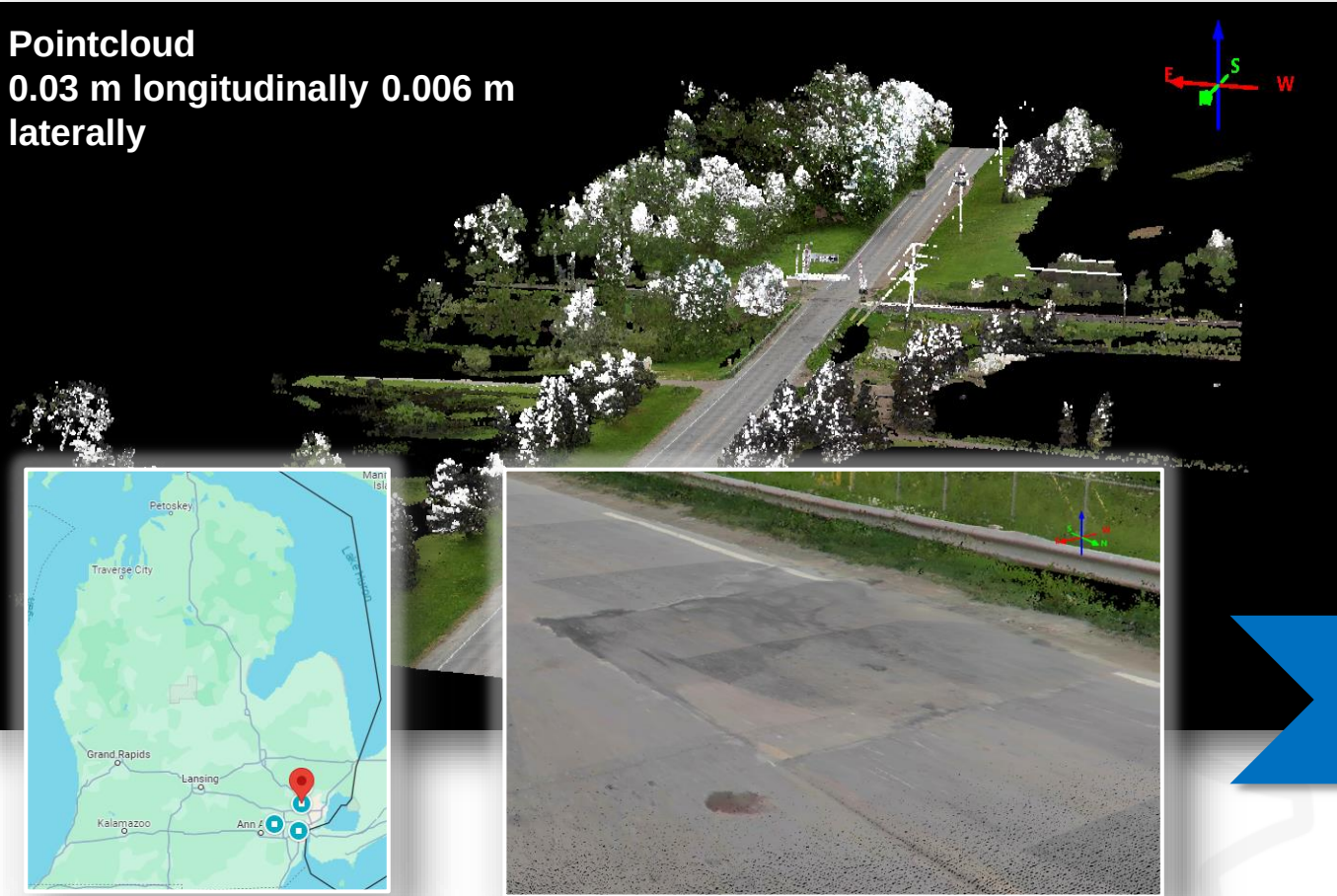


Laser Scanned Point Clouds

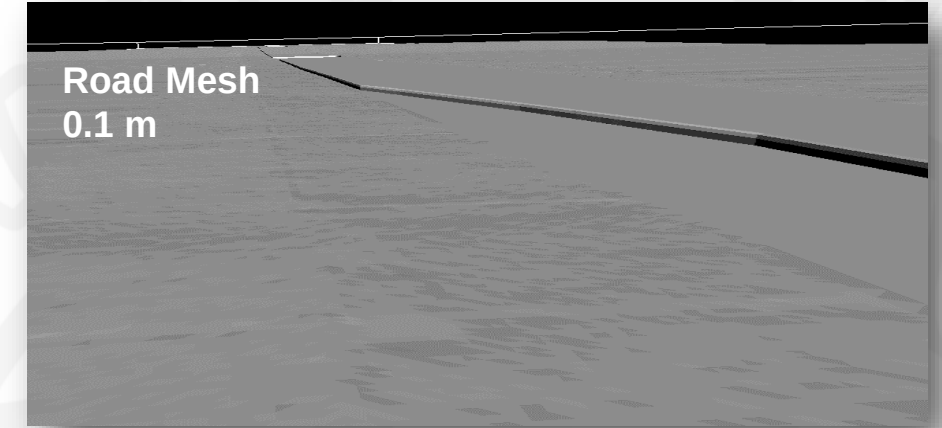


VI-Grade's GridMesh road surface generation process with a resolution of 0.01m

Pointcloud
0.03 m longitudinally 0.006 m
laterally



Road Mesh
0.1 m



Grid Mesh
0.01 m



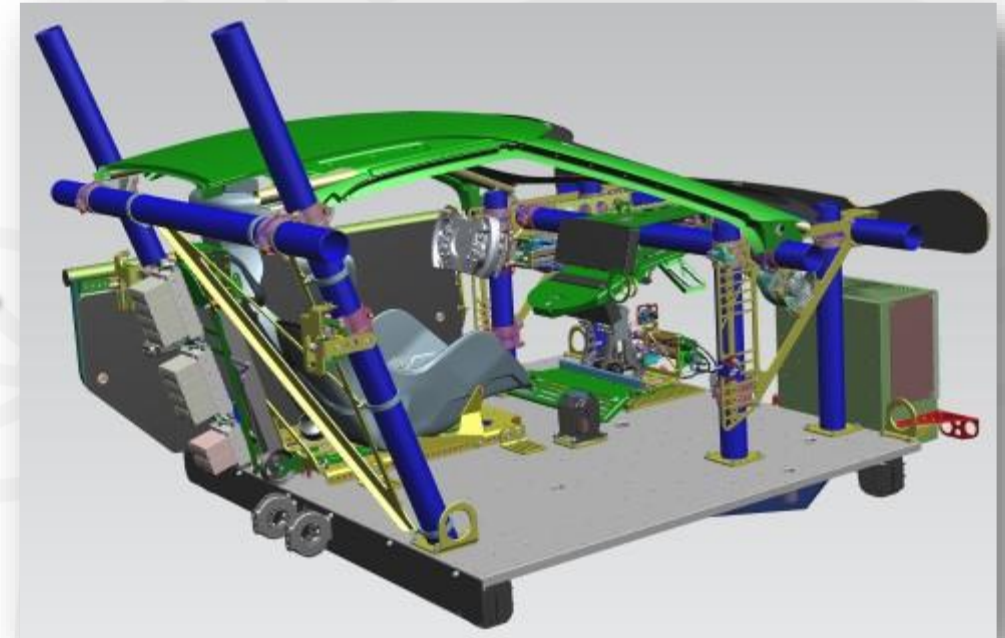
DiM 250 – Modular Cockpit



Due to the wide range of projects and vehicles Multimatic is involved with, a **fully reconfigurable cockpit** have been designed and built.



Example of road car shell from OEMs, tire manufacturers, etc.



Ford GT (Racecar)

Multimatic's modular cockpit

FTire Realtime Simulation



Running on Multimatic & VI-Grade's DiM 250 in Novi Michigan

2 Socket – 16 core CPU, 3.5 GHz Clock

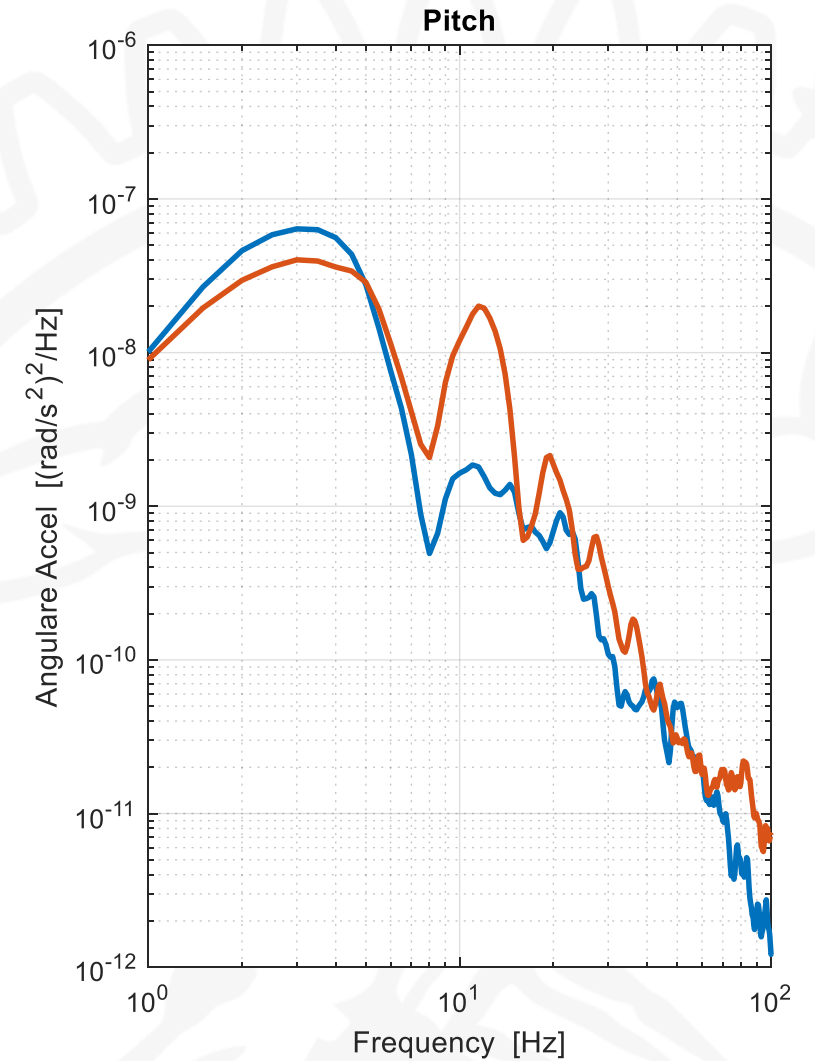
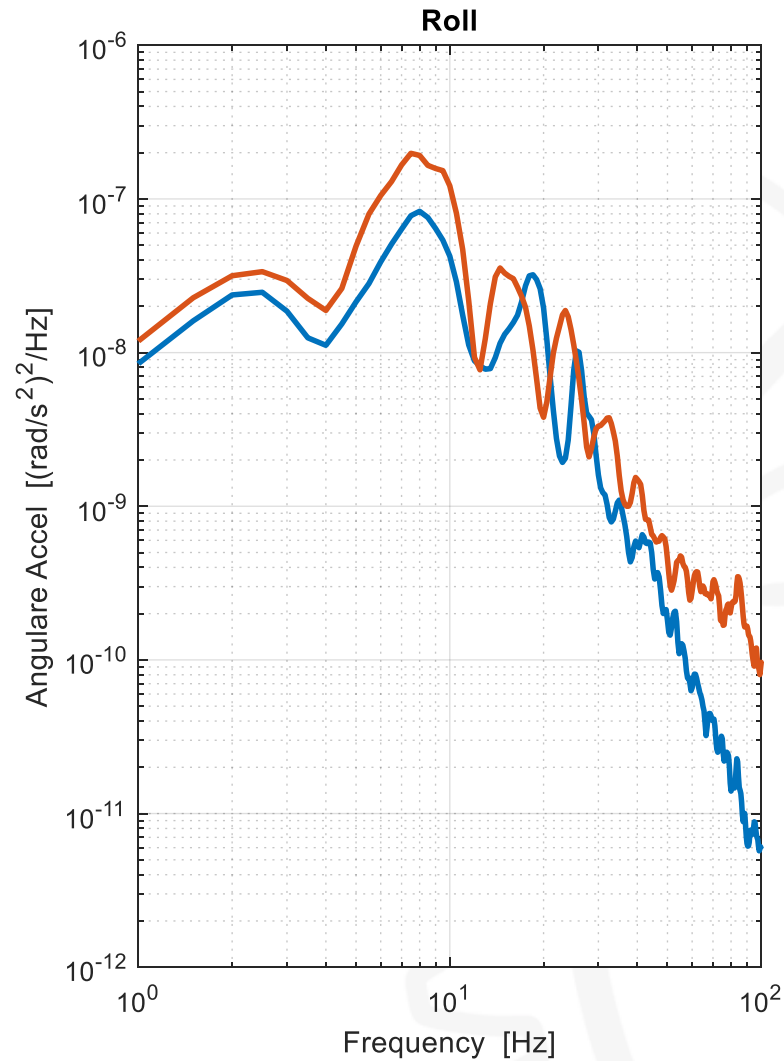
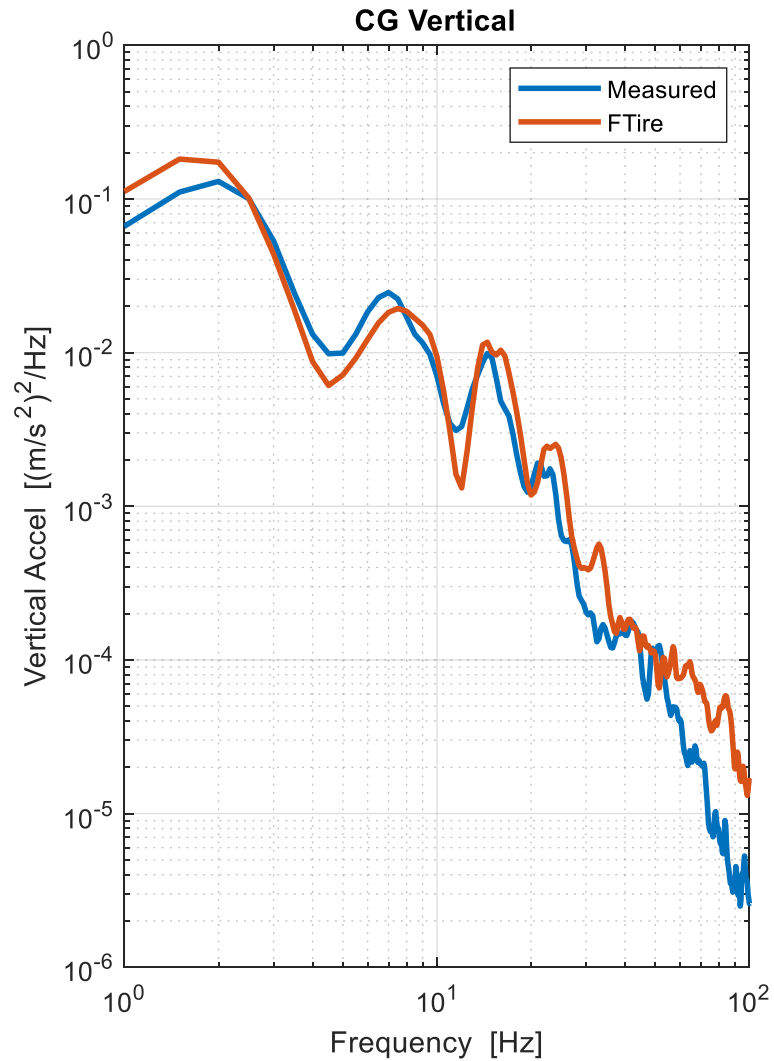
1 millisecond time-step, as normal

Main requirement is one dedicated core per tire

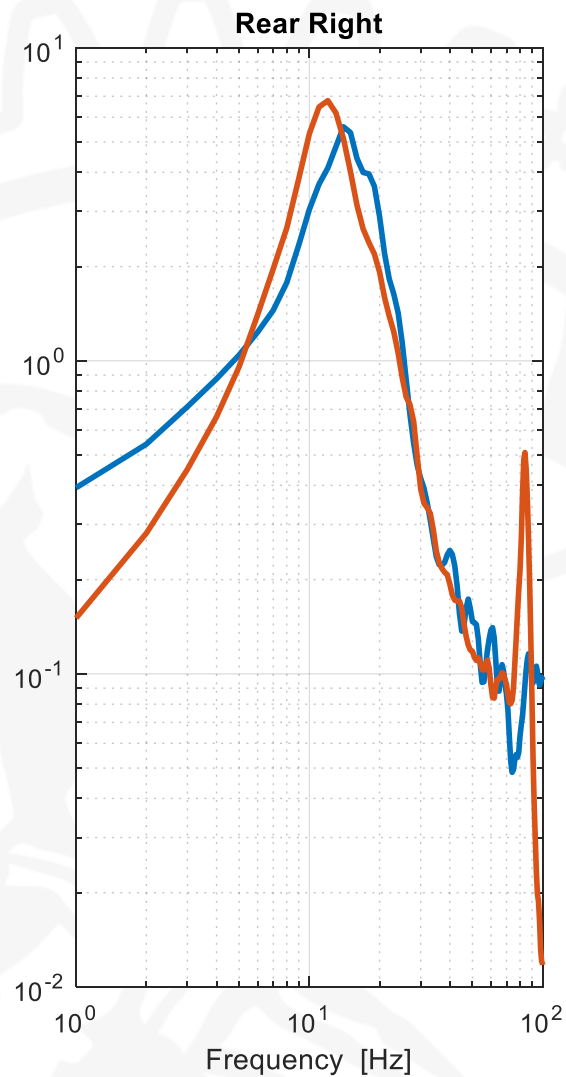
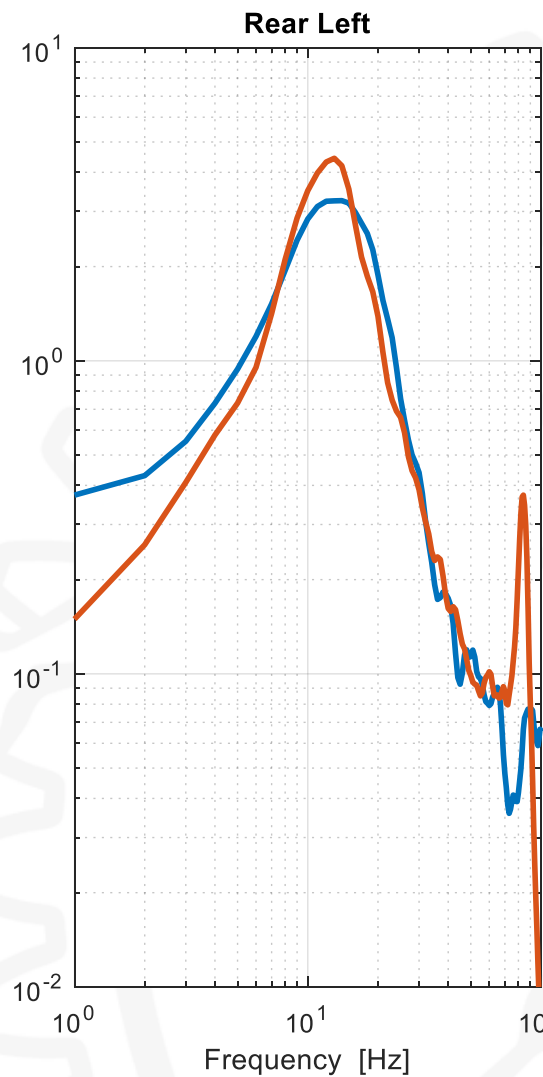
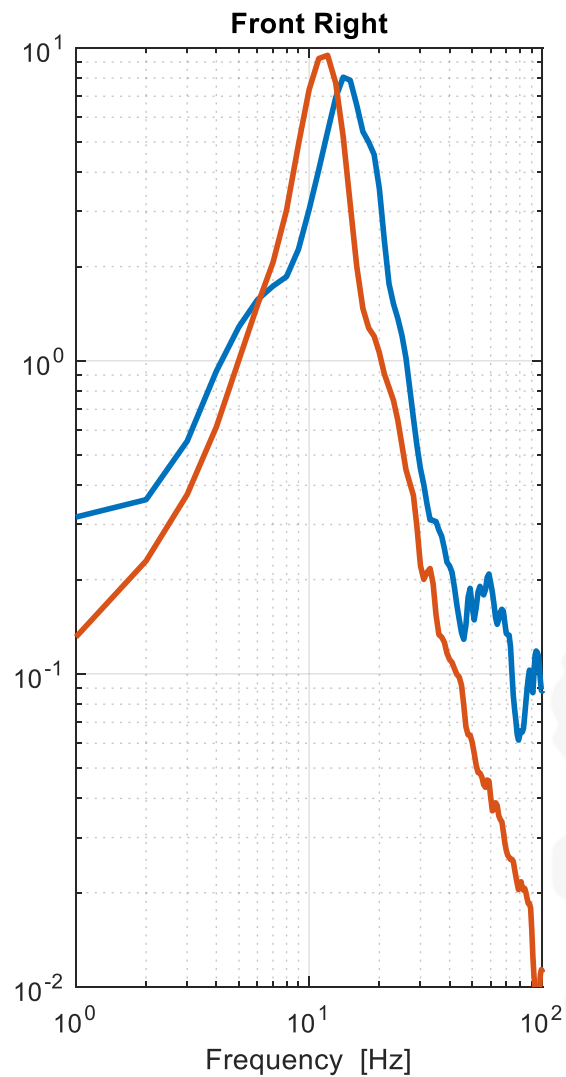
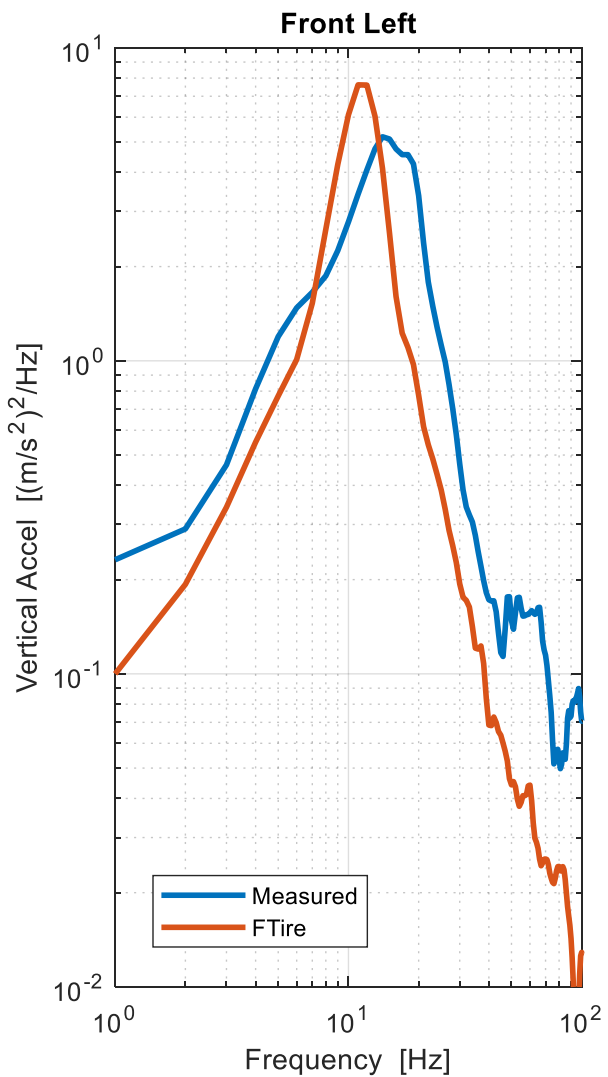
Running FTire in Realtime mode (mode 0)

Reducing the number of tire states from 30 to 15

Body Accelerations



Hub Vertical Accelerations



Next Steps and Future Developments



Development of an FTire model that accurately represents limit handling characteristics with thermal properties

Version 1 - Tradeoff in NVH behaviors acceptable

Version 2 – No tradeoff in NVH characteristics while maintaining limit characteristics

Correlation of FTire contact patch model with Michelin FE tools

Correlation of suspension properties – 4 post test

More vehicle instrumentation – direct hub accelerations, load-cells in the damper load-paths

Greater variety of test surfaces – dependent on tire enveloping behavior

Close the loop on subjective evaluation with the next and final tire submission



Q & A

